# Optional Parts Applicable to S and SE Types

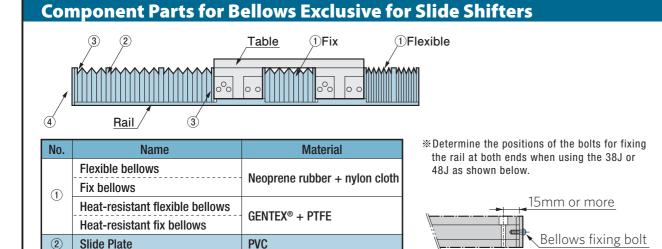


#### Bellows exclusive for slide shifters

The S and SE types incorporate Oiles bearings on the sliding surfaces and have superior foreign matter resistance. It is recommended to use the exclusive bellows if higher resistance is required. A heat-resistant bellows is also available.

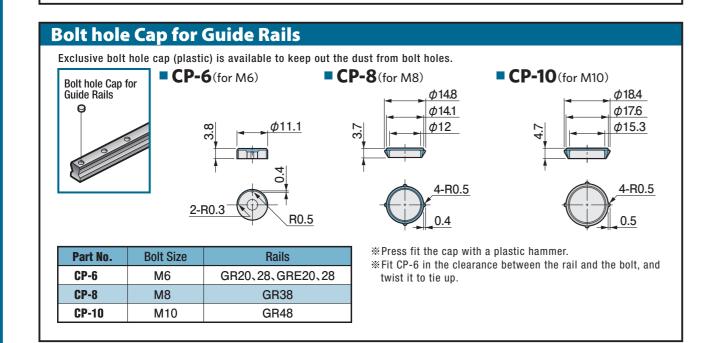
#### Caps for covering up bolt holes on guide rails

Exclusive caps for preventing dust, etc. from entering the bolt holes for mounting the quide rail are available.



SPCC

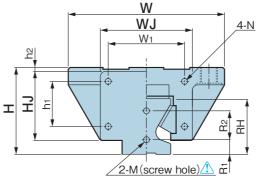
SPCC



## **Product Identification for Exclusive Bellows for Slide Shifters**

## **!** CAUTION

■ Mounting screw hole on the guide rail for bellows is optional.



■ Flexible Bellows

28J - A - 100 - T Part No. └ Put T for heat-resistant bellows - Shortened length of bellows (Lmin) Bellows expansion ratio A or B

■ Fix Bellows

28JK - 140 - T Part No. Put T for heat-resistant bellows Length of fixed bellows

• End plate of 20J sticks out 8mm from the table surface.

Part No.	W×H	Bellows size WJ×HJ	Expansion ratio	Stroke	Expansion ratio	Stroke	h1	h2	W <sub>1</sub>	RH	R <sub>1</sub>	R <sub>2</sub>	N	M	Applicable tables
20J	45×30	52×32	5	under 1100	3.5	1100 or more	14	凸8	37	22	6	10	M3×10	M4×8	STC20 STE20
28J	90×50	60×40	5	under 1100	3.5	1100 or more	26	凹1	44	32	8	18	M3×10	M4×8	STC28 STF28 STE28 STFE28
38J	110×65	80×52	7	under 1300	5.5	1300 or more	36	凹1	58	42	10	24	M4×12	M5×10	STF38
48J	140×82	101×67	10	under 1300	7.5	1300 or more	50	凹3	74	52	12	30	M6×12	M6×10	STF48

### **Calculating formula**

Length of Bellows

(L min=Shortened length, L max=Expanded length)

In case of expansion ratio A
$$L min = \frac{S}{A-1}, L max = L min \times A$$

In case of expansion ratio B 
$$L \ min = \frac{S}{B-1}, L \ max = L \ min \times B$$

Total length of guide rail when using bellows

Using bellows at both ends

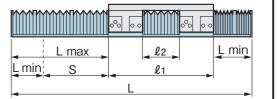
$$L=(L min \times 2) + S + \ell_1$$

Using bellows at one end

#### L=L min+S+ $\ell_1$

In case of standard guid rails, L min dimension needs to be adjusted.

L min=
$$\frac{L-S-\ell_1}{2}$$



S: Stroke

A. B: Expansion ratio of bellows L max: Expanded length of bellows L min: Shortened length of bellows

£1: Table length

 ℓ2: Fix bellows length L: Rail length

#### **Calculation example**

Expression of calculating the bellows length is L min= $\frac{S}{A-1}$ 

L min = 
$$\frac{400}{5-1}$$
 = 100mm

Required rail length L<sub>1</sub>= (L min×2) +S+ $\ell$ <sub>1</sub>

 $L_1 = (100 \times 2) + 400 + 300 = 900 \text{mm}$ 

Bellows length L min when using standard rail length L2 (1000mm)

L min= (1000-400-300) /2=150mm

STF28 Stroke: S=400mm Expansion ratio: A=5 Table length: £1=300mm Fix bellows length: ℓ2=140mm Required rail length: L1 Standard rail length: L2=1000mm

295

Clamp Plate End Plate

## **Durability Test Data / To Prevent Malfunctioning**

## **Durability Test Data**

### S Type

<Testing conditions> Type: STF28 four shift tables GR28-1200 dual-axis

Load: 2,940N {300kgf} Velocity: 0.33m/s {20m/min}

Sliding distance: 1,000km

0 • 0 GR28-1200 Stroke 500mm

<Result>

Wear amount on bushing: 0.032mm

Coefficient of friction: 0.12~0.30

on shaft: 0.006mm

<Result>

Wear amount on liner: 0.025mm

Coefficient of friction: 0.08~0.14

Temperature of friction: 32~42°C

on rail: 0.005mm

### BC Type

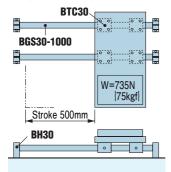
<Testing conditions> Type: BTC30 four shift tables BGS30-1000 dual-axis

Load: 735N {75kgf}

Moment: 323N·m {33kgf·m} Velocity: 0.25m/s {15m/min}

Stroke: 500mm

Sliding distance: 300km (300000 cycles)



<Result>

Wear amount on bushing: 0.023mm

Coefficient of friction: 0.16~0.20

on shaft: 0.012mm

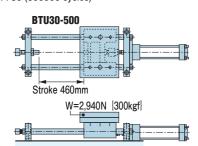
### **BTU Type**

<Testing conditions>

Type: BTU30-500 Load: 2,940N {300kgf Velocity: 0.42m/s {25m/min}

Stroke: 460mm

Sliding distance: 730 (800000 cycles)



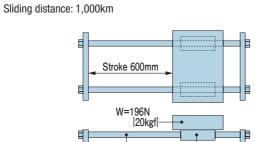
#### **BA** Type

<Testing conditions>

Type: BTCA25-150 one shift table BGS25-800 dual-axis

Load: 196N {20kgf} Velocity: 0.50m/s {30m/min}

Stroke: 600mm



<Result>

Wear amount on bushing: 0.035mm

Coefficient of friction: 0.10~0.25 Temperature of friction: 42~85°C

on shaft: 0.008mm

<Result>

Wear amount on bushing: 0.055mm

Coefficient of friction: 0.20~0.28

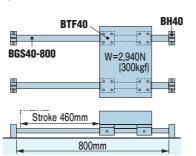
on shaft: 0.008mm

#### BF Type

<Testing conditions> Type: BTF40 four shift tables BGS40-800 dual-axis

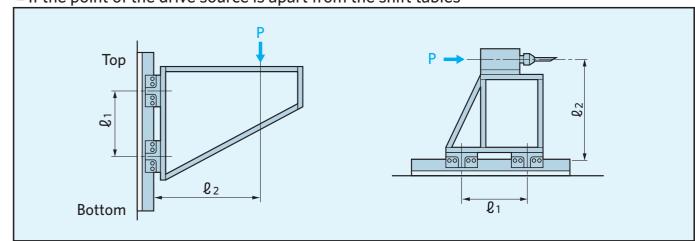
Load: 2,940N {300kgf} Velocity: 0.42m/s {25m/min}

Sliding distance: 1,000km



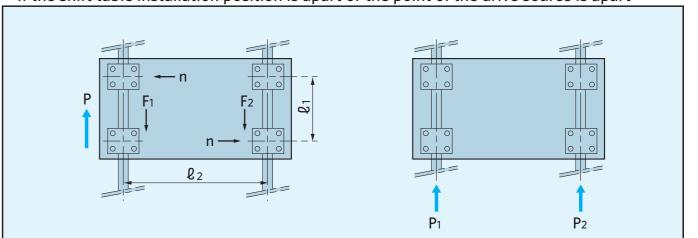
## **To Prevent Malfunctioning**

■ If the point of the drive source is apart from the shift tables



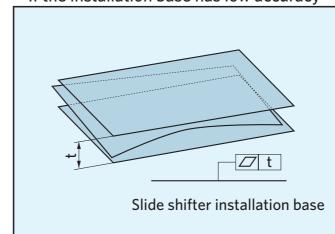
If the position of drive source P is apart from the rail surface by  $\ell_2$ , of  $\ell_2/\ell_1$  exceeds 1.67 when the coefficient of friction  $\mu$  is 0.3, resulting in malfunctioning. Take the allowable moment load into consideration and reduce  $\ell_2/\ell_1$  below 1.5.

#### ■ If the shift table installation position is apart or the point of the drive source is apart

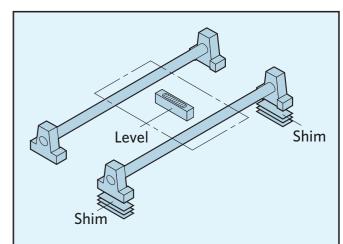


If the \$\mathcal{L}\_2/\mathcal{L}\_1\$ ratio of dual-axis parallel rails is large, the couple of the drive source P and resistance F1 and F2 becomes large and the slide shifter works improperly. Reduce £2/£1 below 3. As the point of the drive source becomes apart from the center, the condition becomes worse. Synchronize the drive source with P1 and P2 if  $\ell 2/\ell 1$  is inevitably larger than 3 for reasons of the structure.

#### If the installation base has low accuracy



Do not select the S type if the parallelism t exceeds 0.3.



Select the B type if the parallelism t exceeds 0.3. Insert shims under the shaft holders to adjust them.

After adjustment, check the parallelism with a level, straight edge,